

Performance evaluation of a Semivolatile Aerosol Dichotomous Sampler (SADS) for Exposure Assessment: impact of design issues.

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Abstract. Aerosols of semivolatile organic compounds (SVOCs) pose significant health risks to workers in various occupational settings. Measuring human exposure to these aerosols requires a separate assessment of the contribution of particles and gases, which is not resolved by existing sampling techniques. Here, we investigate experimentally the performance of the Semivolatile Aerosol Dichotomous Sampler (SADS), proposed in previous studies, for sampling monodisperse liquid particles with aerodynamic diameters between 0.15 and 4.5 μm , corresponding to workplace aerosols. The measured sampling performances are compared to their theoretical counterparts computed by computational fluid dynamics. The effects of leakage rate, repeatability of the assembly, imprecision of the actually machined nozzle diameters and SADS parts misalignment are examined. The SADS assembly is found easily leaky, but consequences on sampling can be overcome when a prior leak test with leakage rate below 4 $\text{Pa}\cdot\text{s}^{-1}$ is passed. Variation of nozzle diameters in the range (-4.5 %, +3.7 %) with respect to nominal values affects marginally (<3 %) aerosol transmission efficiency, but sampling performance is little reproducible during successive SADS assemblies (CV=22.1 % for wall losses). Theoretically unpredicted large (40-46 %) wall losses are measured for particles larger than 2 μm , located mostly (80 %) on the external walls of the collection nozzle. Assembly repeatability issues and simulations of SADS parts misalignment effect by CFD suggest that these undesirable particle deposits are due to the mechanical backlashes of the assembly. Thus, the current design does not guarantee a nozzle misalignment of less than 5 % of the acceleration nozzle diameter, and other important geometric parameters are not further constrained. The promising theoretical sampling performance of the SADS for SVOCs aerosol larger than 1 μm thus falls short of expectations due to mechanical design issues that can be improved before possible field use.

1 Introduction

Semivolatile organic compounds (SVOCs) represent a significant subgroup of volatile organic compounds (VOCs), and their presence in the environment raises concerns due to their association with carcinogenic, mutagenic, and reprotoxic effects (Raffy et al., 2018). One of the defining features of SVOCs is their ability to exist simultaneously in both vapour and particle phases, making their sampling and analysis a complex task. There are varying definitions of SVOCs, with the U.S. Environmental Protection Agency (Technical Overview of Volatile Organic Compounds, 2020) proposing a classification based on boiling points (240-380 °C at atmospheric

36 pressure), while the standard EN 13936 defines them according to their saturation vapour pressure (ranging from
37 0.001 to 100 Pa at room temperature).

38 In occupational settings, SVOCs can be encountered in diverse forms, such as metalworking fluid (MWF) mists,
39 phthalates, pesticides, acrylamides, machining fluids, exhaust gases from diesel engines, and more. Exposure to
40 these pollutants in the workplace can pose health risks to workers, depending on their chemical nature and the
41 extent of exposure. Understanding the health implications requires accurate measurement of both vapour and
42 particle phases, as their behaviour and effects can differ when inhaled in the respiratory system. Notably, the
43 vapour phase's absorption in the respiratory tract is influenced by the SVOC's solubility, while particle deposition
44 in the lung is governed by their aerodynamic diameter (Volckens, 2003). Additionally, direct adsorption of SVOCs
45 on the skin can lead to absorption into the body (Roberts et al., 2009). Thus, it becomes essential to separately
46 quantify the concentrations of each phase that constitutes a semivolatile aerosol.

47 Various sampling techniques have been proposed to evaluate semivolatile aerosol concentrations, including
48 filtration methods, thermodesorption tubes, cyclones, electrostatic precipitators, and multiple "filter + adsorbent"
49 devices. However, these methods often suffer from the issue of evaporation of the particulate phase during
50 sampling, leading to biased measurements. Filtration methods, for instance, have been found to underestimate
51 particle concentrations due to continued evaporation from the filter during sampling (Park et al., 2015; Raynor et
52 al., 2000; Simpson, 2003; Simpson et al., 2000; Volckens et al., 2010). Other techniques that do not instantaneously
53 separate particles and vapour also face the problem of evaporation during sampling (Raynor and Leith, 1999; Leith
54 et al., 2010; Lillienberg et al., 2008; Wlaschitz and Höflinger, 2007; Sutter et al., 2010; Kim and Raynor, 2010a).
55 As of now, no applicable model exists to theoretically calculate the evaporation of a semivolatile aerosol during
56 workplace air sampling, which hampers the use of these techniques.

57 An alternative approach is the Virtual Impactor (VI) principle, inspired by classical inertial impactors with
58 collection plates (Loo and Cork, 1988; Marple and Chien, 1980). The VI is employed for size classification of
59 particles based on their aerodynamic diameter. In 2009, the Semivolatile Aerosol Dichotomous Sampler (SADS),
60 a novel variant of the VI dedicated to SVOCs, was proposed by Kim and Raynor (2009) and raised great hopes
61 for this application.

62 In its original version, the SADS features an inverted flow configuration between the major and minor flows,
63 resulting in 86 % of the total air being directed into the collection nozzle, while the remaining 14 % is suctioned
64 perpendicular to the acceleration nozzle axis (Figure 2). The aerosol is sampled through a 4 mm inlet orifice and
65 accelerated through a convergent shape called the "acceleration nozzle", which narrows to a 0.8 mm orifice
66 diameter. In the separation space, inertial particles are directed into the collection nozzle, while low inertial
67 particles and vapours follow both the major and minor flows. The collection nozzle, located 1.2 mm from the
68 acceleration nozzle, has a diameter of 1.1 mm, and filters and adsorbent beds are placed at each outlet (major and
69 minor flow). The SADS operates at a total sampling flow rate of 2.1 L.min⁻¹, split into 1.8 L.min⁻¹ at the major
70 flow outlet and 0.3 L.min⁻¹ at the minor flow outlet (split ratio of 0.143).

71 The SADS was further optimized numerically and tested both in the laboratory and in the field by its designers
72 (Kim and Raynor, 2010a, b; Kim et al., 2014). The optimized version is characterized essentially by a revised split
73 ratio of 0.1 with a total sampling flow rate of 2 L.min⁻¹ and with a length of the separation space reduced to
74 0.48 mm instead of 1.2 mm. The angle of the acceleration nozzle was also changed from 19° to 45° between the
75 2009 and 2010 versions. Despite these modifications, the overall mechanical design of the SADS has not changed

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77 between these two versions. The overall shape of the device is that of a 37 mm cassette, and it is made up of two
78 parts that fit together via a cylindrical bearing surface. The parts are held together by 2 screws. Sealing is ensured
79 by an O-ring between the two parts, pressed together by the two screws.

80 Thus, the work of Kim et al. led to the creation of the SADS concept and revealed its interest in the sampling of
81 semi-volatile aerosols. However, many questions remain before SADS can be considered sufficiently mature for
82 widespread use as a portable sampling device for SVOC aerosols.

83 Firstly, the sampling performance of the device was not evaluated in detail for particles with aerodynamic
84 diameters greater than 1 μm , as the initial device was not designed for this. However, for field use, the evaluation
85 of the performance of the SADS for particles above 1 μm is especially important because workplace SVOCs
86 aerosols showed a presence of particles with diameters up to 10 μm (Cooper et al., 1996; Park et al., 2009). Since
87 the metric of SVOC exposure is mass concentration, and the mass carried by particles increases with the cube of
88 their diameter, sampling errors on the most inertial particles generate biases in exposure measurements that are far
89 more problematic than sampling errors on sub-micron particles. Optimization work by Kim et al. focused on
90 reducing the cut-off diameter of the device around an aerodynamic diameter of 0.7 μm , but the impact on super-
91 micron particle sampling was not assessed. Subsequent tests on real aerosols revealed significant deposits in the
92 device that had not been anticipated by the theoretical study, and the exact origin of these deposits is still unknown
93 (26.5 % of wall losses for an aerosol with MMAD of 2.17 μm in Kim and Raynor (2010b) and separate evaluations
94 by NIOH, Norway and Fraunhofer ITEM showed similarly high deposition ratios (Olsen et al., 2013).

95 Secondly, it's important to emphasize the absence of published documentation or feedback regarding the
96 mechanical realization and the necessary operating procedures for obtaining measurements in line with theoretical
97 performance for the SADS. It is well known that the details of mechanical design and manufacture have as much
98 to do with impactor performance as the theoretical design: sealing, nozzle alignment (Loo and Cork, 1988),
99 geometric assembly tolerances - these are all necessary qualities which are the consequence of a suitable
100 mechanical design. So far, the SADS design proposed by Kim et al. has not been studied from these aspects, and
101 it is possible that a more definitive version of the SADS will require a review of its overall mechanical design,
102 without modification of the interior volume, which is perhaps optimal. Summarily, the authors who worked on the
103 SADS have neither published documentation related to these aspects nor investigated them in previously published
104 articles. In particular, it is doubtful whether the device as shown in Kim and Raynor (2010b) is leak-tight, with
105 only two diametrically opposed clamping points. Also, in both versions, the proposed design does not seem to
106 guarantee a precise control of nozzle spacing and alignment during assembly (limited guides and ground seats).

107 Finally, it is not certain that the optimized version proposed in 2010 is really optimal for sampling semi-volatile
108 aerosols encountered at workplaces, for various reasons. Firstly, from a methodological point of view, the
109 optimization carried out is based on a Computational Fluid Dynamics (CFD) model, which does not appear to be
110 sufficiently accomplished to provide enough accurate results for the optimization approach to have been
111 conclusive, as exposed in Belut et al. (2022). This is notably illustrated by significant differences between CFD
112 predictions and measurements, for both the airflow and the aerosol phase (modelled pressure drop on the major
113 flow side is more than twice the measured value for Kim and Raynor (2009), measured and simulated particles
114 separation efficiencies depart by as much as 30 % in Kim and Raynor (2009) and by as much as 53 % in Kim and
115 Raynor (2010b), simulations report almost no particle deposition but experimental evaluations found important
116 wall losses: 26.5 % for an aerosol with MMAD of 2.17 μm in Kim and Raynor (2010a) and separate evaluations

118 by NIOH, Norway and Fraunhofer ITEM showed similarly high deposition ratios (Olsen et al., 2013). Secondly,
119 the 2010 version has a much steeper inlet convergent angle (acceleration nozzle) than the 2009 version, which
120 increases the probability of undesirable wall loss for the most inertial aerosols (Belut et al., 2022).

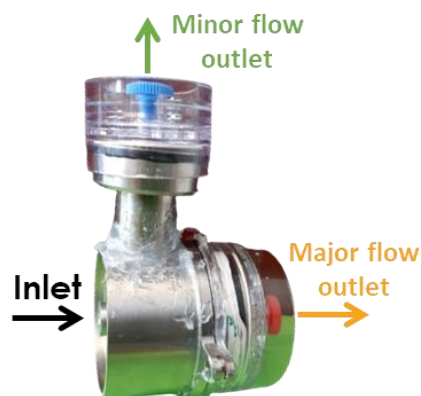
121 In this context, the present article;

- 122 - Conducts an experimental investigation into the size-resolved sampling performances of the SADS on
123 liquid SVOCs particles within an aerodynamic diameter range of 0.15-4.5 μm , i.e., extending beyond
124 previous studies;
- 125 - Identifies practical issues related to the design, manufacturing and operation of the SADS as proposed in
126 Kim and Raynor (2009, 2010b);
- 127 - Investigates the effect of small variations in SADS nozzle diameters due to inevitable random
128 manufacturing uncertainties;
- 129 - Details the localization of wall losses in the device, as mentioned in prior research, and identifies their
130 cause.

131 In doing so, our aim is to suggest improvement targets for future realization of the SADS, a small step to obtain a
132 device suitable for the dichotomic measurement of particles and vapours composing SVOCs aerosols at
133 workplaces. We shall base our study on the 2009 version of the SADS, because of the smaller cut-off diameter
134 and also considering that the issues related to the overall design of the SADS are common to both versions.

135 To reach our objectives, five 2009 SADS prototypes were constructed and their sampling behaviour was
136 characterized, using monodisperse liquid aerosols of various sizes. After evaluating the leakage resistance of the
137 assembly, and its consequences on wall deposition, the actual sampling performances are compared to their
138 theoretical counterparts computed by Belut et al. (2022). Origin of discrepancies are examined in terms of
139 influence of the actually machined nozzle diameter and of the repeatability of the SADS assembly. An analysis of
140 the distribution of deposits within the SADS is then used to estimate the likely cause of deposits in the device. The
141 results are then discussed to propose improvement targets for the realization of the SADS, in terms of design and
142 assembly.

143 Where necessary, CFD simulation results are used to support the observations. The approach of Belut et al. (2022)
144 is then used for this purpose, including systematic calculation verification steps.



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146 Figure 1 : Photography of a SADS with schematic airflows directions.

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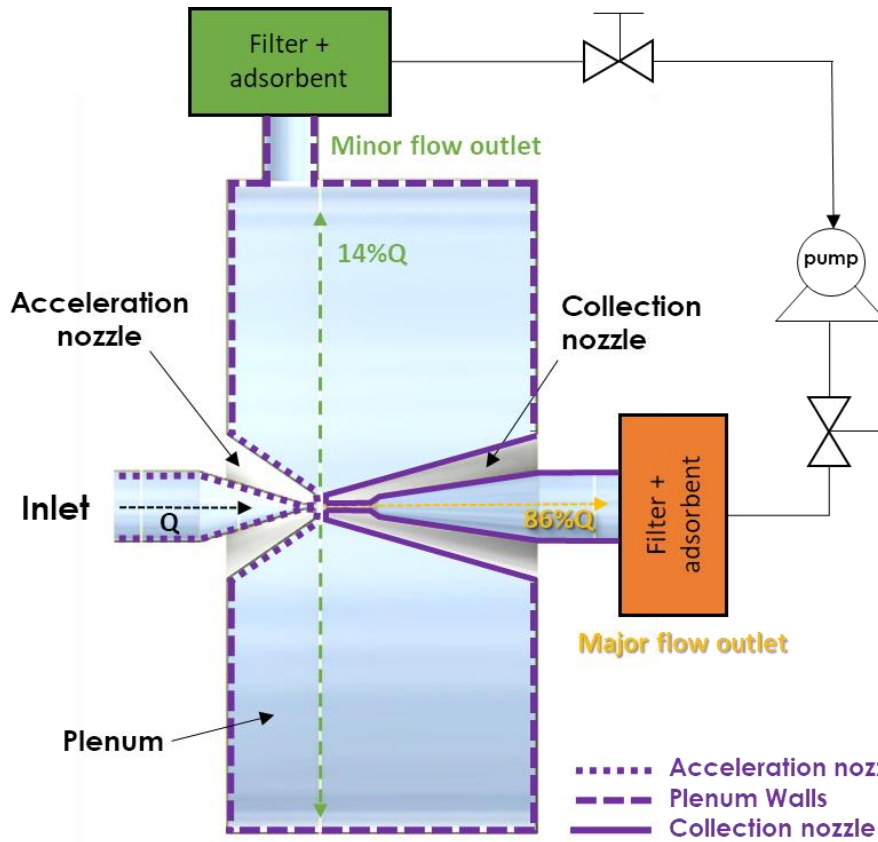
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175 Figure 2; Schematic diagram of the Semi-volatile Aerosol Dichotomous Sampler (SADS). The diagram illustrates
 176 the positioning of the acceleration and collection nozzles, the division of the inlet airflow into a major (86% of the
 177 inlet flow) and a minor flow outlet (14% of the inlet flow). Each outlet is equipped with a filter and adsorbent for
 178 effective aerosol sampling. The different types of walls, used to determine the mass of particles collected on, are
 179 depicted by various dashed lines: Acceleration nozzle Walls, Plenum Walls, Collection nozzle Walls.

180 **2. Principle of the particle-vapour dichotomous sampling in the SADS and definitions**

181 The SADS is derived from the VI principle with an inverted split ratio between the major flow and the minor flow.
 182 The intended behaviour is as follows: the mixture of air, SVOCs vapours and SVOCs aerosol particles is drawn
 183 into the device through its inlet (Figure 2). This mixture is first accelerated by a convergent nozzle and reaches the
 184 separation gap of the device. In the separation gap, a small fraction of the flow is deflected laterally and directed
 185 to a first outlet, while a large fraction continues forward through a divergent collection nozzle. Because of their
 186 inertia, the particles should preferentially follow the main flow, while the concentration of the vapours remains
 187 unaffected by the separation of the flow. With ideal sizing, the minor flow is then free of particles, allowing the
 188 concentration of the vapor phase of the sampled SVOCs to be measured. Subtracting this vapor contribution from

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201 the mass of SVOCs collected with the main stream then allows the concentration of SVOC aerosol particles only
202 to be retrieved. With an ideal design, the SADS then allows dichotomous analysis of airborne SVOCs.

203
204 For a given aerodynamic diameter of particles d_{ae} , the particles transmission efficiency $\eta_p(d_{ae})$ to the particle
205 major flow outlet is defined as the ratio of particle mass collected at the major flow outlet to the total particulate
206 sampled mass of particles with the same diameter (Eq. 1):

$$207 \eta_p(d_{ae}) = \frac{m_{major}(d_{ae})}{m_{inlet}(d_{ae})} \quad (1)$$

208 Similarly, $\eta_v(d_{ae})$ is the particles transmission ratio to the particle minor flow outlet, defined by the ratio of the
209 particle mass collected at the minor flow outlet to the total particulate sampled mass (Eq. 2), for a given particle
210 size.

$$211 \eta_v(d_{ae}) = \frac{m_{minor}(d_{ae})}{m_{inlet}(d_{ae})} \quad (2)$$

212 Finally, we defined a particles deposition ratio $\eta_d(d_{ae})$ that correspond to the ratio of the mass deposited on the
213 inner wall of the SADS to the total particulate sampled mass, for a given particle size (Eq. 3):

$$214 \eta_d(d_{ae}) = \frac{m_{dep}(d_{ae})}{m_{inlet}(d_{ae})} \quad (3)$$

215 In ideal working conditions of the SADS as a gas-particle separator, we expect η_d and η_v to be zero while $\eta_p=1$. It
216 is the details of the device's geometric dimensions and the choice of minor and major flow rates that determine the
217 device's theoretical separation performance (Loo and Cork, 1988; Marple and Chien, 1980). In the present article,
218 these choices are assumed to be theoretically optimal, and we study only the effects of certain design and
219 manufacturing details on the device's ability to actually achieve its theoretical performance. Hence, minor and
220 major flows are set constant at their theoretical optimum as specified.

221
222 In these conditions, the theoretical performances of the SADS in terms of η_p , η_v , and η_d has been extensively
223 studied numerically by (Belut et al., 2022), their work highlighting the main factors influencing the
224 representativeness of the CFD modelling of similar devices. In described operating conditions, they indicate that
225 $\eta_v = 0$ and $\eta_p > 98\%$ for d_{ae} in [0.9 - 20] μm (perfect separation). We may introduce a d_{50} cut-off diameter as
226 the aerodynamic diameter of particles below which η_v is equal to half its maximum value of 0.143 (corresponding
227 to no separation, in this case η_v equals the gas split ratio). Results from (Belut et al., 2022) indicate that the
228 theoretical d_{50} of the SADS is 0.44 μm , i.e. much smaller than most of SVOCs aerosol diameters at the workplace.
229 The SADS is hence in theory perfectly suitable for field use, where most of SVOCs aerosol diameters are above
230 (Cooper et al., 1996; Park et al., 2009). However, these performances are theoretical, whence the necessity of
231 finding the conditions under which this is valid in actual realization of the SADS.

232 3 Material and methods

233 3.1 Aerosol generation

234 To assess the performance of the Semivolatile Aerosol Dichotomous Sampler (SADS), aerosols were generated
235 using a specialized equipment called the Bench for Organic Aerosol (BOA), as depicted in Figure 3. The BOA is
236 a vertical wind tunnel designed to operate with controlled airflow velocities ranging from 0.1 to 0.5 m.s⁻¹ and

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a supprimé: If this is verified, the particle and vapour concentration of SVOC in the sampled air can be easily obtained by collecting the total mass (vapour+particles) of SVOC at each outlet (minor and major) and by knowing that the minor outlet sample corresponds to vapours only (it is particle-free). The major outlet sample corresponds to particles and vapours, but the vapour contribution, known from the minor flow, can be subtracted to yield the particle concentration. However, this principle is valid only if =0 and =1 for the sampled aerosol size range, whence the necessity of finding the conditions under which this is valid in actual realization of the SADS.

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a supprimé: η_p

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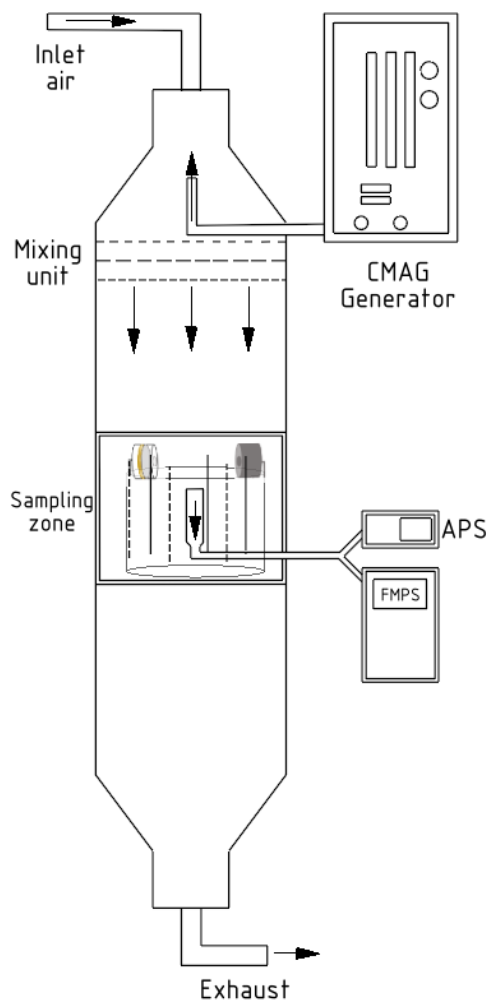
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288 humidity levels between 10 to 90 %RH. Aerosols were introduced counter current at the head of the tunnel to
289 ensure thorough mixing with the airflow. To achieve spatially homogeneous mixtures at the sampling zone, the
290 air/aerosol mixture passed through a series of diaphragms with different meshes. The SADS prototypes, along
291 with an isokinetic nozzle connected to online instrument measurements, were placed on a crown support in the
292 sampling zone.

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294 **Figure 3: Schematic drawing of the Bench for Organic Aerosol (BOA) generation device.**

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295 To maintain consistent experimental conditions, room temperature was set to 21 °C, atmospheric pressure was
296 measured at 1018 ± 10 hPa, relative humidity was regulated at 20 %RH, and airflow velocity was fixed at
297 $0.2 \text{ m}\cdot\text{s}^{-1}$. The airflow inside the tunnel was considered isothermal, incompressible, and turbulent, with a Reynolds
298 number based on the tunnel size of approximately 4×10^3 .

299 The BOA was carefully calibrated to meet the requirements of the standard NF EN 13205-2:2014. Spatial
300 homogeneity of velocities in the sampling section was confirmed, with the standard deviation below 1 % over the
301 entire sampling zone. Similarly, the spatial homogeneity of particle concentration demonstrated standard deviation
302 values below 10 %.

306 3.2 Aerosol generator

307 To produce aerosols for testing the SADS prototypes, a specialized Condensation Monodisperse Aerosol Generator
308 (CMAG - TSI 3475) based on Sinclair-La Mer principle (Sinclair and La Mer, 1949) was employed. This specific
309 generator condenses heated vapours of diethyhexyl sebacate (DEHS) homogeneously on thin particles of sodium
310 chloride, referred to as nuclei, to form monodispersed liquid particles. The size of these particles ranges from 1 to
311 8 µm, depending on the selected generation conditions. Originally designed for use with diethyhexyl sebacate
312 (DEHS) and NaCl, the CMAG was modified to accommodate the use of glycerol and fluorescein (Steiner et al.,
313 2017). This modification was necessary as DEHS is not water-soluble, making the analysis of particles collected
314 on filters or internal walls of the sampler challenging and less sensitive. By replacing DEHS with glycerol and
315 NaCl with fluorescein, water-soluble droplets were generated, and fluorescein could be quantified at a very low
316 concentration (i.e., LoQ < 1 ng.L⁻¹ within water extract).

317 The aerosol production process within the CMAG initiates with the nebulization of a water solution, specifically
318 composed of 0.8 g.L⁻¹ fluorescein and 5 g.L⁻¹ sodium hydroxide in pure water, within an atomizer. This step is
319 succeeded by the drying of the droplets in a diffusion dryer. Following the diffusion dryer, small nuclei, constituted
320 of a blend of fluorescein and sodium hydroxide, were generated. These nuclei, serving as condensation nuclei,
321 exhibited sizes ranging from 10 to 100 nm. These nuclei were then exposed to a saturated vapour of glycerol
322 downstream of the saturator. The resulting mixture of glycerol vapour and nuclei was directed to a re-heater and
323 subsequently cooled down in a condensation chimney to produce the monodispersed aerosol.

324 It is important to note that the size of the generated particles could be adjusted by modifying the temperature of
325 the saturator or the number concentration of nuclei. For this study, aerosols with mass median aerodynamic
326 diameters (MMAD) of circa 0.15, 2, 3, and 4.5 µm were produced and used for the experiments.

327 3.3 Aerosol characterization

328 Characteristics of the generated aerosols were measured continuously during the generation process. Aerodynamic
329 particle sizes and geometric standard deviations (GSD) were measured using a TSI Aerodynamic Particle Sizer
330 (APS 3321) associated with an aerosol diluter (TSI 3302 A) for particles ranging from 0.5 to 20 µm. For particles
331 ranging from 0.056 to 0.560 µm, a TSI Fast Mobility Spectrometer (FMPS – 3091) was used. The FMPS apparatus
332 measures a mobility diameter that was converted in this study in an aerodynamic diameter using the following
333 equations, considering that all particles generated during this study were spherical;

$$334 d_{ae} = d_m \left(\frac{Cu(d_m) \rho_p}{Cu(d_{ae}) \rho_0} \right)^{1/2} \quad (4)$$

335 Where d_m is the particle mobility diameter, d_{ae} the aerodynamic diameter, Cu the Cunningham correction factor
336 calculated with the appropriate diameter, ρ_0 the reference density (1000 kg.m⁻³) and ρ_p and the real density of the
337 particle (kg.m⁻³).

338 The particle density exhibits variability between nuclei and condensed glycerol particles. Based on the initial
339 composition of the fluorescein solution utilized for generating nuclei, the density of the nuclei was determined to
340 be 1720 kg.m⁻³ after total desiccation. In contrast, the density of the condensed particles is approximated to the
341 density of pure glycerol, given the negligible mass of the nuclei compared to the mass of glycerol that condenses
342 on them. Consequently, particles with diameters of 2 µm and above are considered to possess a density of
343 approximately 1260 kg.m⁻³.

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$$d_m = d_{ev} x \frac{Cu(d_m)}{Cu(d_{ev})},$$
$$d_{ae} = d_{ev} \left(\frac{Cu(d_{ev}) \rho_p}{Cu(d_{ae}) \rho_0 x} \right)^{1/2}$$

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a supprimé: , d_{ev} called the equivalent volume diameter, : $d_{ev} = \sqrt[3]{\frac{6V_p}{\pi}}$ with V_p the particles volume, $Cu(d_m)$ the Cunningham slip correction factor based on electrical mobility diameter,

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377 To further enhance the relevance of this study, the physical diameter of the particles is approximated by the
 378 measured aerodynamic diameter, considering the spherical nature of the particles. This approximation facilitates
 379 the conversion of the number-based particle size distribution into a mass-based particle size distribution, a
 380 parameter of greater significance for our research objectives. Following the conversion from a number-based to a
 381 mass-based particle size distribution, we proceeded to calculate the mass median aerodynamic diameter (MMAD).
 382 This parameter serves as a valuable metric, providing a comprehensive characterization of the aerosol particles in
 383 our investigation.

384 The measurement of aerosol characteristic by APS and FMPS apparatus allow modulating the particle diameter
 385 produced by the CMAG and verifying the stability of the aerosol concentration during the experiment. Averages
 386 are shown in [Table 1](#). Note that the aerosol with a MMAD of 0.16 μm exhibited a high GSD of 1.56, which does
 387 not meet the monodisperse criteria with a GSD < 1.2. Specifically for this aerosol, the particles measured were
 388 actually nuclei generated by removing the glycerol from the CMAG. Consequently, without glycerol condensation
 389 on their surfaces, their diameters could not be homogenized. In summary, we typically measured the particle size
 390 distribution of nuclei generated before condensing glycerol on them to produce micron-monodispersed particles

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391 **Table 1: Averaged particle size distributions of the test aerosols (N=3 ± SD).**

Aerosol reference diameter (μm)	MMAD ± SD (μm)	GSD ± SD
0.15	0.16 ± 0.05	1.56 ± 0.02
2	2.04 ± 0.15	1.16 ± 0.03
3	3.17 ± 0.21	1.14 ± 0.01
4.5	4.70 ± 0.12	1.10 ± 0.02

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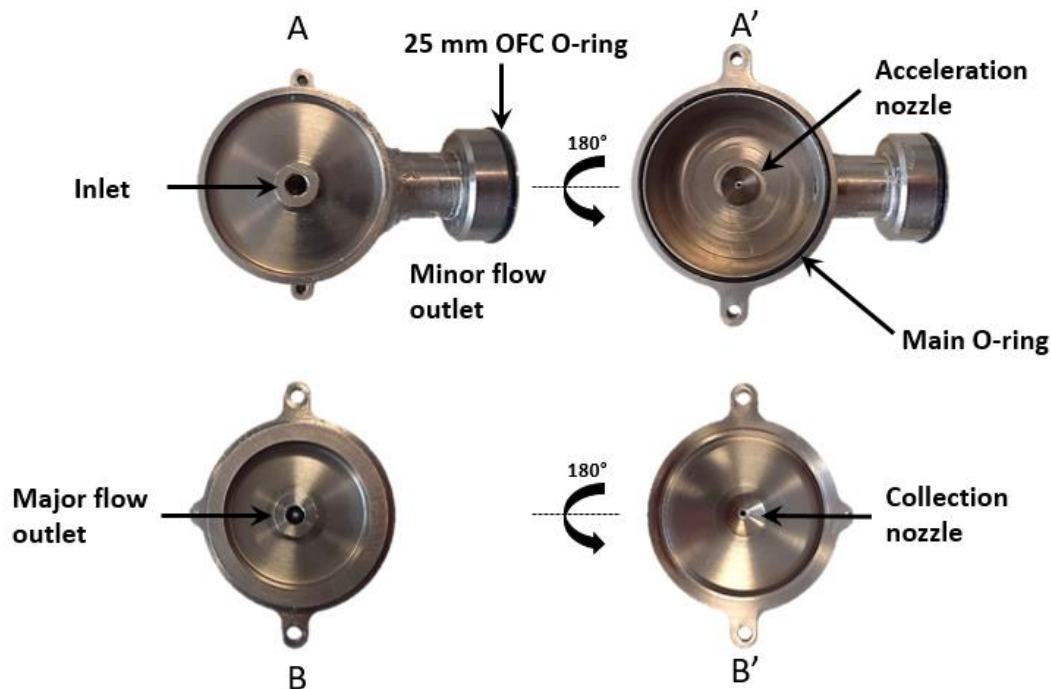
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392 The stability of aerosol concentration in the sampling zone was verified, with a mass concentration deviation
 393 below 6 % across experiments.

394 3.4 SADS prototypes

395 The STAMI, Norway, had five titanium SADS Kim & Raynor (2009) prototypes. Photographs of the SADS
 396 prototype components are presented in Figure 4, illustrating the acceleration nozzle and the collection nozzle,
 397 along with the connection of a cassette to the SADS sampler. The SADS prototypes investigated in this study were
 398 manufactured by a precision mechanics workshop, resulting in slight deviations in their nozzle diameters compared
 399 to the reference dimensions proposed by Kim & Raynor (2009). These deviations were attributed to the inherent
 400 tolerance of the manufacturing process. Specifically, one of the prototypes (SADS R) exhibited nozzle diameters
 401 identical to those of the 2009 version, serving as the reference case for comparisons. The remaining prototypes
 402 (SADS 1, SADS 2, SADS 3, and SADS 4) showed minor differences in their nozzle diameters relative to the
 403 reference, as detailed in
 404 Table 2. The deviations, both absolute and relative, are provided for both the acceleration and collection nozzles.
 405 These dimensions were measured using scanning electron microscopy (SEM) for accuracy. The maximum
 406 deviation of nozzle diameter with respect to the reference dimensions of SADS R (Kim and Raynor (2009)
 407 dimensions) was +0.03 mm (+3.7 %) for the acceleration nozzle and -0.05 mm (-4.5 %) for the collection nozzle.

a supprimé: manufacture d by a precision mechanics workshop. Each prototype consisted of two main pieces: the acceleration part and the collection part, as shown in Fig. 3. For all SADS prototypes, the actually machined nozzle diameters were measured using scanning electron microscopy (SEM). One of the manufactured SADS presented nozzle diameters exactly equal to those of the 2009 version of the SADS proposed by Kim & Raynor (2009): it was denoted SADS R and considered as the reference case for comparisons. The other SADS prototypes (SADS 1, SADS 2, SADS 3, and SADS 4) exhibited slight deviations in their nozzle diameters with respect to the SADS R, due to the inherent tolerance of the manufacturing process. The measured diameters of the acceleration and collection nozzles for each SADS prototype are presented in ¶ Table 2.



448

449 **Figure 4:** Photographs of the SADS prototype, consisting of two main components - the acceleration nozzle (A') and
 450 the collection nozzle (B'). The upper left photo (A) shows the inlet side of the acceleration nozzle, while the lower left
 451 photo (B) displays the outlet side of the collection nozzle. On the outlet side of the collection nozzle (B), a 37 mm
 452 cassette is easily connected to the SADS sampler.

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Table 2: Nozzles diameters measured by SEM.

SADS name	Acceleration nozzle [mm] <i>(absolute difference with reference / relative difference with reference)</i>	Collection nozzle [mm] <i>(absolute difference with reference / relative difference with reference)</i>	Ratio Collection nozzle / Acceleration nozzle
SADS 1	0.83 <i>(+0.03 / 3.7 %)</i>	1.05 <i>(-0.05 / -4.5 %)</i>	1.27
SADS 2	0.82 <i>(+0.02 / 2.5 %)</i>	1.06 <i>(-0.04 / -3.6 %)</i>	1.29
SADS 3	0.798 <i>(-0.002 / 0.3 %)</i>	1.11 <i>(+0.01 / + 0.9 %)</i>	1.39
SADS 4	0.818 <i>(+0.018 / 2.2 %)</i>	1.11 <i>(+0.01 / + 0.9 %)</i>	1.36
SADS R	0.8 <i>(+0.00 / 0 %)</i>	1.10 <i>(+0.00 / 0 %)</i>	1.38

455 3.5 Sampling procedure

456 The performance evaluation of the five SADS prototypes was conducted simultaneously in the sampling zone of
 457 the Bench for Organic Aerosol (BOA) (Figure 3). Prior to testing, each prototype was meticulously cleaned using
 458 ethanol and pure water. The samplers were equipped with Whatman Quartz Microfiber Filters (QMA) placed into
 459 37 mm and 25 mm Open Face Cassettes (OFC) and connected at the major outlet and the minor outlet (Figure 2),
 460 respectively. The flow rates at the major flow outlet (1.8 L.min⁻¹) and the minor flow outlet (0.3 L.min⁻¹),

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465 corresponding to a total inlet flow rate of 2.1 L.min⁻¹, were precisely controlled using flow meters (Gilian
466 Gilibrator-2).

467 3.6 Fluorescence analysis

468 After each generation test, the sampling procedure for fluorescence analysis was carried out. The Whatman Quartz
469 Microfiber Filters (QMA) contained in the 37 mm and 25 mm Open Face Cassettes (OFC) were extracted
470 separately and analysed for fluorescence content. Each filter in the CFC and OFC was inserted into independent
471 vials. A volume of 2 to 8 mL of the extraction solution, consisting of ultrapure water with a concentration of 5 g.L⁻¹
472 of NaOH, was added to the vial to dissolve the collected droplets of glycerol and their fluorescein/sodium
473 hydroxide nuclei. The walls of the CFC were also washed with the extraction solution (pure water basified with
474 5 g.L⁻¹ of NaOH), and the resulting volume was combined with the one in the vial containing the CFC filter. After
475 20 minutes of mechanical shaking, the extracts were filtered through a PTFE syringe filter with a pore size of about
476 0.2 µm to prevent any disruption of the fluorescence measurement.

477 Wall deposition inside the SADS was determined by using 2 mL of the extraction solution to wash each wall of
478 the SADS separately. Three different extracts were obtained: one from the acceleration nozzle wall (carried particle
479 mass m_{dep_a}), one from the collection nozzle wall (carried particle mass m_{dep_c}), and one from the plenum wall
480 (carried particle mass m_{dep_p}), (Figure 2).

481 The extracts were then analysed for fluorescence using a portable ESElog Fluorescence Detector (Qiagen,
482 Germany), with an excitation wavelength of 485 nm and an emission wavelength of 520 nm. The linear range of
483 the ESElog Fluorescence Detector defined the lower (LLOQ) and upper (ULOQ) limits of quantification, which
484 covered the concentrations encountered in this work (LLOQ = 0.33 ng.L⁻¹, ULOQ = 4×10⁴ ng.L⁻¹).

485 For each tested aerosol aerodynamic diameter d_a , the total sampled mass m_{inlet} is evaluated as the sum of sampled
486 masses:

$$487 m_{inlet} = m_{dep_a} + m_{dep_c} + m_{dep_p} + m_{major} + m_{minor} \quad (5)$$

488 and the fractional deposition ratio $\eta_d(d_{ae})$ is computed as:

$$489 \eta_d(d_{ae}) = (m_{dep_a} + m_{dep_c} + m_{dep_p}) / m_{inlet} \quad (6)$$

490 and local deposition ratios at the acceleration nozzle (η_{d_a}), collection nozzle (η_{d_c}) and in the plenum (η_{d_p}) are
491 respectively computed from:

$$492 \eta_{d_a} = m_{dep_a} / m_{inlet} \quad (7)$$

$$493 \eta_{d_c} = m_{dep_c} / m_{inlet} \quad (8)$$

$$494 \eta_{d_p} = m_{dep_p} / m_{inlet} \quad (9)$$

$$495 \text{so that } \eta_d = \eta_{d_a} + \eta_{d_c} + \eta_{d_p} \quad (10)$$

496 Because monodispersed aerosols are used, the masses evoked in this paragraph are all linearly related to the amount
497 of fluorescent dye that they carry. Hence, deposition ratios are directly computed from the measured masses of
498 fluorescent dye.

499 3.7 Mass balance verification

500 A mass balance verification step was adopted to verify that the protocol allowed recovering all particles sampled
501 by the SADS. The SADS prototypes and 37-mm Closed Face Cassettes (CFCs) were arranged alternately on the

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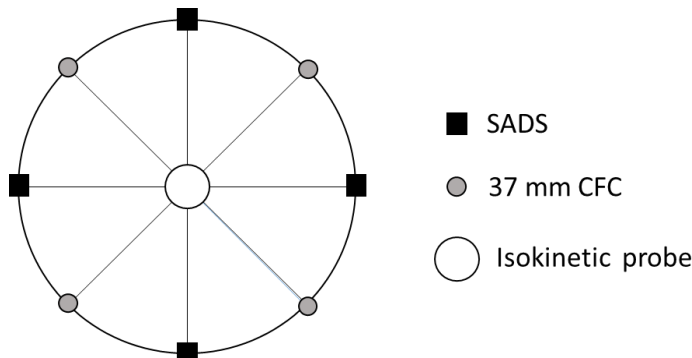
510 crown support (Figure 5). The 37-mm CFCs served as reference samplers, enabling the determination of the total
511 mass of particles collected, in comparison to the particle mass measured by the SADS prototypes.

512 All SADS and 37-mm CFCs presented a standardized 4 mm aerosol-sampling orifice and operated at the same
513 sampling flow rate of 2.1 L.min⁻¹. This standardization ensured that the total mass collected inside the 37-mm
514 CFCs corresponded to m_{inlet} , the total mass sampled by the SADS, which includes the contribution of both the
515 Open Face Cassettes (OFC) placed at the minor and major flow outlets and the wall deposits. By comparing m_{inlet}^i ,
516 the total particle mass measured by the SADS i , with the mean reference mass measured by the 37-mm CFCs
517 ($\overline{m_{CFC}}$), we introduce the mass balance ratio of SADS i as:

518 $Mb_{SADS}^i = m_{inlet}^i / \overline{m_{CFC}}$ (11)

519
520 This mass balance makes it possible to assess the overall efficiency of the protocol used to measure the distribution
521 of particles collected by SADS, between deposits and major and minor outlets.

522 Additionally, the use of the 37-mm CFCs allows assessing the spatial homogeneity of the aerosol distribution
523 within the sampling zone. No significant spatial variation was observed across different positions of the CFCs (CV
524 < 5 %). With this assurance of spatial homogeneity, the individual SADS mass balance (Mb_{SADS}) was calculated
525 using equation 5 for each of the SADS prototypes.



526
527 **Figure 5:** Schematic representation of the positions of the samplers on the crown support.

528 3.8 Leak evaluation

529 Any form of leakage is known to compromise the sampling performances of aerosol samplers such as SADS, by
530 disrupting the airflow and path lines within the nozzles and separation zone. Experimental tests were carried out
531 to examine the leakage resistance of the proposed SADS assembly, to observe the effects of leaks and to determine
532 an acceptable leakage limit for the SADS.

533 As the SADS operates under depression, a leakage test was performed using a digital pressure calibrator (DPC -
534 FSM AG) set to a depression of -4000 Pa, equivalent to the operating pressure of the system. Following a
535 stabilization period, the DPC's internal pump was deactivated, and the pressure was continuously measured to
536 determine the leakage rate (LR) in Pa.s⁻¹ (Eq. 5). Three levels of air tightness were defined: low, medium, and
537 high, corresponding to LR values of $LR \geq 13 \text{ Pa.s}^{-1}$, $4 \text{ Pa.s}^{-1} \leq LR < 13 \text{ Pa.s}^{-1}$, and $LR < 4 \text{ Pa.s}^{-1}$, respectively.

538 $LR = \frac{|P_0 - P_f|}{\Delta t}$ (12)

539 Where P_0 and P_f represent the pressures at $t = 0$ s and at the final time, respectively, and Δt is the duration of the
540 leak test.

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547 3.9 Supporting CFD Model

548 The CFD modelling approach employed in this study to support observations is documented in detail in Belut et
549 al. (2022). Simulations are conducted using ANSYS FLUENT V.19.3 software. After due examination of the most
550 influent modelling and physical factors affecting the significance of results, a 2D axisymmetric reduction of the
551 inner volume of the SADS is used to perform simulations. A low-Reynolds realizable k- ϵ turbulence model is used
552 to model the incompressible airflow, with a free-inlet boundary condition at the entrance of the SADS, following
553 the guidelines of Belut et al. (2022). Aerosol particle fates are computed through a Lagrangian tracking of their
554 centre of mass, taking into account turbulent dispersion and using a free-inlet boundary condition at the inlet.
555 External forces acting on particles are reduced to drag force, including rarefaction effect. Impaction and
556 interception phenomena are taken into account for wall losses, particles being assumed trapped when hitting a wall
557 (consistent with the liquid nature of present aerosols). An extensive verification of computations with respect to
558 grid size, numerical resolution tolerances and number of used aerosol trajectories was performed, exactly as
559 exposed in Belut et al. (2022). For further insights into the model's design and its applicability to the SADS,
560 interested readers are encouraged to refer to the aforementioned study. [Following Belut et al. \(2022\), simulation
561 results are realistic, within the calculated uncertainties, unless one of the following occurs: 1\) the SADS walls are
562 not smooth, 2\) there is a difference between the actual and simulated geometry, 3\) residual turbulence exists at the
563 SADS inlet \(with a Kolmogorov timescale much greater than the aerodynamic response time of the particles, which
564 does not correspond to normal ambient conditions\).](#)

565 4 Results and discussion

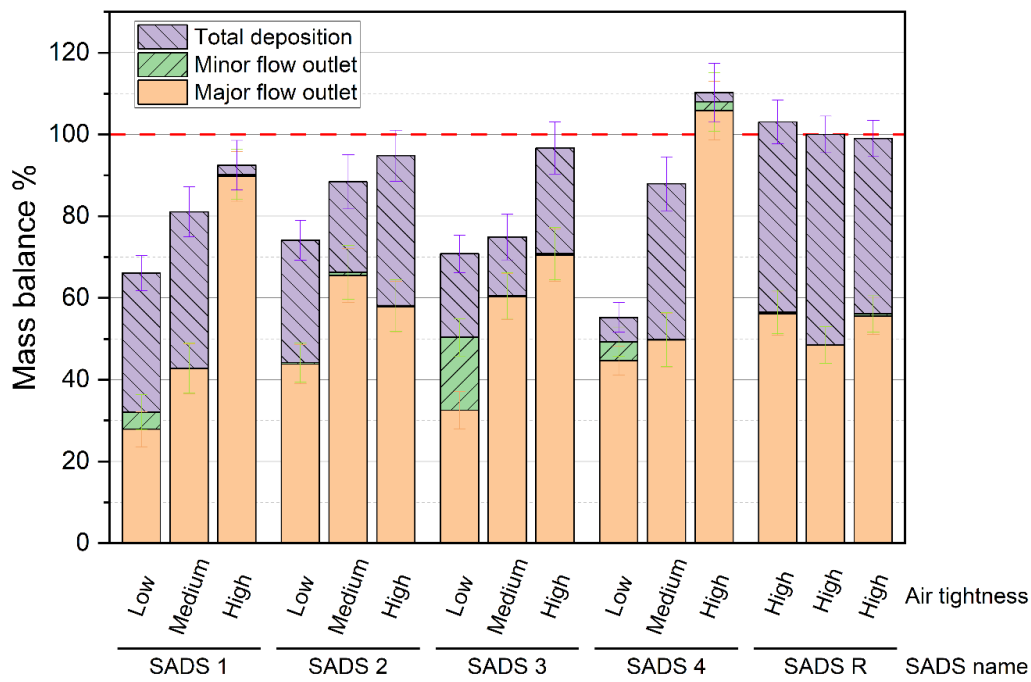
566 4.1 Leaks effects

567 The air_tightness tests conducted revealed significant leakages, primarily occurring at the O-ring seal between parts
568 A and B ([Figure 2](#)) of the SADS prototypes. Additional leaks were also identified at the connection points between
569 the Open Face Cassette (OFC) and the major and minor outlets of the SADS. Notably, SADS 1, 2, 3, and 4
570 displayed varying levels of air_tightness during the tests, with the exception of SADS R, which consistently
571 exhibited high air_tightness across all tests. Fig. 6 illustrates the evolution of mass balance and mass distributions
572 for the five SADS prototypes with varying levels of air_tightness.

573 The aerosol generated in the three tests was monodisperse with a size distribution centred on a MMAD of
574 $3.11 \pm 0.21 \mu\text{m}$, with a GSD of 1.14 ± 0.03 and a particle number concentration of $10124 \pm 320 \text{ pcl.cm}^{-3}$.

575 Tests on the leaks effects on the performance of the SADS were also conducted with aerosols having MMAD of
576 2 and 4.5 μm . The results and conclusions were consistent with the distributions presented, indicating that the
577 outcomes converge towards those presented in the following section.

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581

Figure 6: Mass balance in the SADS prototypes in function of the air tightness level. Error bars represent the standard deviation calculated on five replicates for each condition.

582 Low air tightness led to a substantial decrease in the mass balance of SADS 1, 2, 3, and 4 (Figure 6), with mass
583 balances reaching $66 \pm 6.2\%$, $74 \pm 6.8\%$, $70 \pm 6.58\%$, and $55 \pm 7.59\%$, respectively. Only when a high level of
584 air tightness was achieved could a mass balance of 90% or higher be attained for all prototypes.

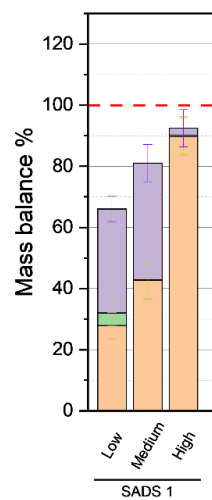
585 Moreover, low air tightness resulted in an undesirable increase in the mass fraction collected at the minor flow
586 outlet. Presumably, leaks disturbed the airflow in the separation space, leading to the deviation of larger particles
587 to the minor flow outlet than theoretically expected.

588 Globally, leaks not only influenced the total amount of particles collected within the SADS but also affected the
589 particles transmission ratio to the major and minor outlets, which make results from leaky SADS unreliable.

590 A systematic leak test is then mandatory before using the SADS for sampling purposes. To ensure reliable and
591 accurate results, the SADS should only be considered suitable for sampling when the leakage rate (LR) is below 4
592 $\text{Pa}\cdot\text{s}^{-1}$. Keeping the leakage rate within this acceptable limit will help maintain the integrity of the SADS and
593 improve the reliability of the data collected during sampling operations. For further development, we also
594 recommend revising the design of future realization of the SADS to guarantee its air tightness.

595 Due to difficulties in maintaining a constant level of air tightness for SADS 3 and 4 throughout all experimental
596 tests, further experimentation with these two prototypes was not conducted.

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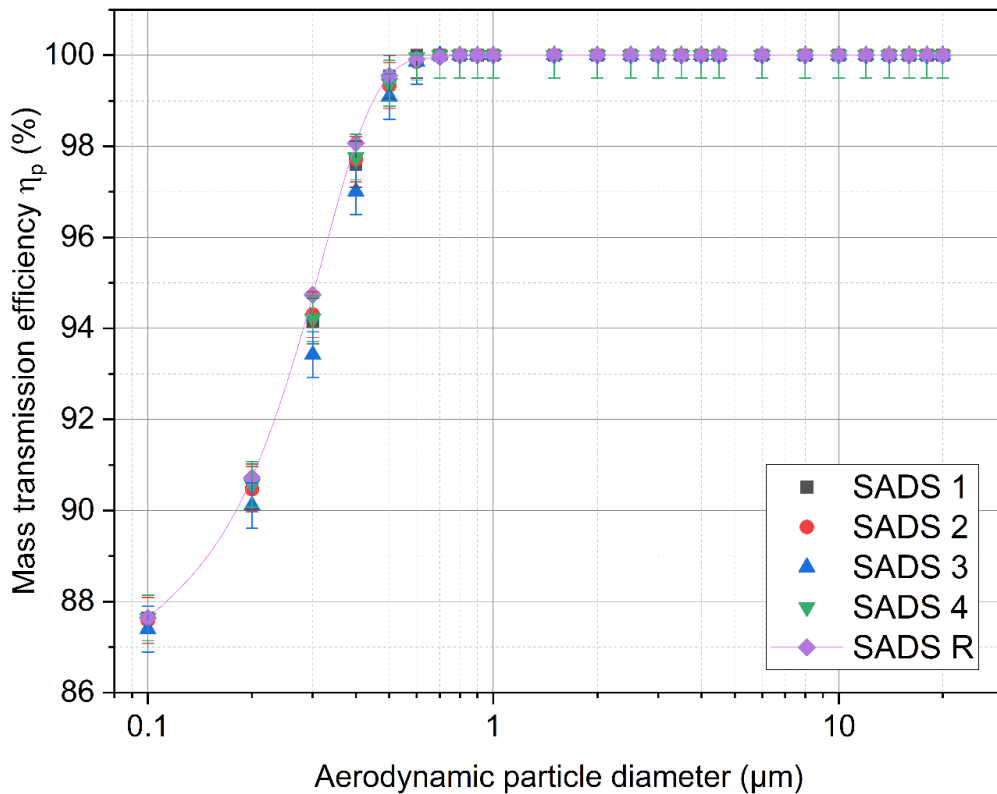


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601 **4.2 Theoretical effect of actually manufactured nozzle diameters**

602 Before comparing theoretical and actual performances of manufactured SADS, the effect of the lack of precision
603 on the actually manufactured nozzle diameters is examined from a theoretical point of view, using the CFD model
604 with the measured nozzle diameters of SADS 1, 2, 3, 4 and R. The numerical model computed the η_p curves for
605 each SADS prototype across a range of aerodynamic particle diameters from 0.1 to 20 μm , and the results are
606 presented in [Figure 7](#). Error bars correspond to one standard deviation of values arising from turbulent dispersion.



607 [Figure 7](#): Evolution of CFD mass transmission of the SADS prototypes, accounting for variation in nozzle
608 diameters.
609

610 The theoretical η_p curves calculated by the numerical model are similar for all SADS prototypes ([Figure 7](#)).
611 SADS 3 exhibits the maximum difference compared to the reference SADS R transmission efficiency curve, but
612 this difference remains below 3 % for all diameters. Overall, the variations observed in the nozzle diameters
613 actually machined are not expected to lead to radically different sampling performance between the different
614 prototypes.

615 These results are consistent with findings from a previous study on a VI by Marple & Chien, 1980a, who observed
616 that increased ratios between nozzle diameters led to increased wall deposition, above the recommended value of
617 1.33 (and thus to decreased transmission ratios). From

618 [Table 2](#), we indeed see that SADS 3 exhibits the largest nozzle diameters ratio (1.39) of the prototypes. All ratios
619 are, however, below 1.49 as recommended by Marple & Chien, 1980a.

a supprimé: Fig. 6

Mass transmission efficiency η_p (%)

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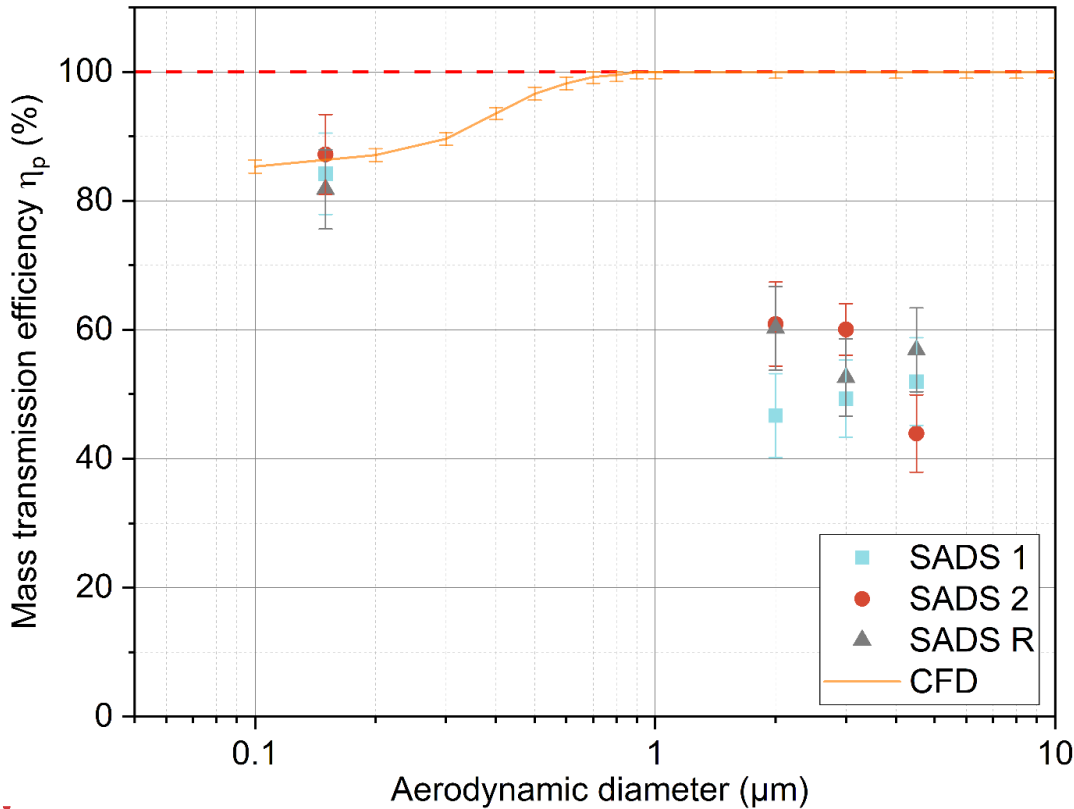
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627 **4.3 Actual vs. theoretical particle transmission efficiency**

628 **Figure 8** compares the theoretical and actually measured η_p curves of the 3 airtight SADS.



629
630 **Figure 8:** Comparison of theoretical CFD transmission efficiency η_p (SADS R) with experimental results for SADS 1,
631 SADS 2, and SADS R. Error bars represent one standard deviation for CFD simulations due to turbulent dispersion
632 and measuring uncertainty for experimental data.

633 For aerosols with a reference diameter of 0.15 μm , numerical predictions are in accordance with experimental
634 tests, with transmission efficiencies to the major and minor flow outlets close to the ratio between the major and
635 the minor flow ($\eta_p = 86 \pm 0.58\%$ for the model and $\eta_p = 81.7 \pm 6\%$ experimentally). This corresponds to the
636 expected behaviour of low inertial particles that are not separated by the SADS. We shall see, however, in the
637 following section that a substantial fraction of these particles is actually deposited experimentally, in contradiction
638 with theoretical results.

639 For the inertial particles tested with nominal diameters 2, 3 and 4.5 μm , η_p is measured as always less than about
640 60 %, whereas 100 % is theoretically expected for the SADS in free-sampling situation. The origin of this
641 difference is examined first by considering the particles deposition ratio in the next sections.

642 **4.4 Fate of inertial particles and repeatability issues**

643 To illustrate the origin of the unexpectedly low transmission efficiency of inertial particles in the device, the
644 distribution of all measured η_p , η_v and η_d for 3 repetitions of the experiment and for the 3 airtight prototypes is
645 shown on **Figure 9**, and

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Mass transmission efficiency η_p (%)

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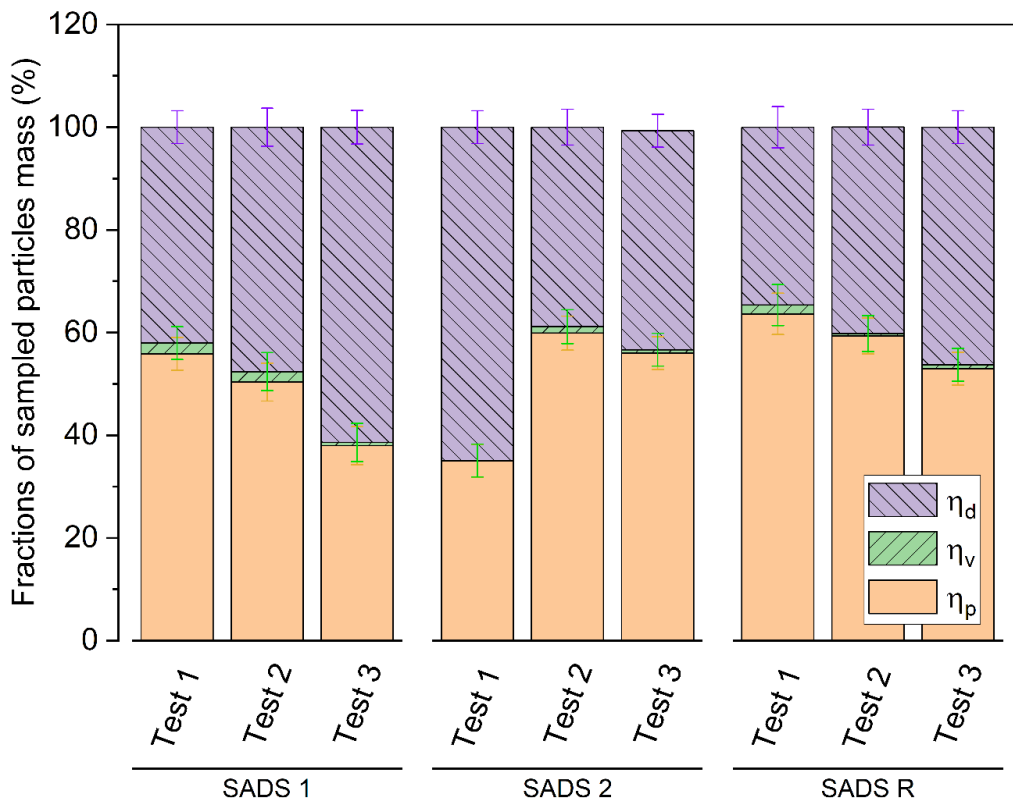
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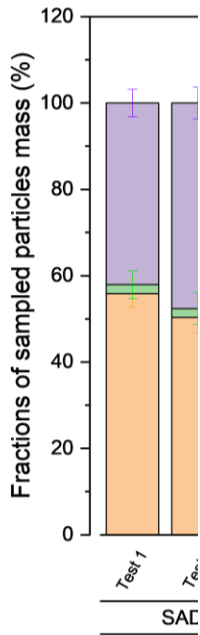
650 **Table 3.** Only results for the 4.5 μm particles are shown here for brevity. For these particles, we observe that the
 651 low transmission efficiency η_p is attributable to large ($46.6 \pm 5.4 \%$) wall losses (η_d), and not to the misdirection
 652 of particles to the minor outlet. These deposits are not theoretically explained, even if we take into account the
 653 lack of precision of machined nozzle diameters (Figure 8), and we can note that they apparently vary randomly
 654 across repetitions with a large coefficient of variation for η_d (22.1 %). These variations are then likely to be
 655 attributable to the assembly process of the SADS, since other influencing parameters were monitored and
 656 controlled (flow rates, aerosol particle sizes, homogeneity of concentrations in the BAO, SADS leakage rate
 657 similarity of sampled masses).

a supprimé: Table 3

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658

659 **Figure 9:** Distribution of the fate of inertial particles with a reference diameter of 4.5 μm in multiple repetitions, for
 660 SADS 1, SADS 2, and SADS R at High Airtightness Level. Error bars represent the measuring uncertainty associated
 661 with the data points."

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662

668 **Table 3: Transmission ratio to the major and minor outlets and deposition ratio for SADS 1, 2 and R.**

SADS name	Tests	η_p	η_v	η_d
		[%]	[%]	[%]
SADS 1	Test 1	55.9	2.08	42.1
	Test 2	50.3	2.04	47.6
	Test 3	38.0	1.26	61.4
	Average	48.1	1.79	50.4
	SD	9.2	0.46	10.0
SADS 2	Test 1	35.0	0.07	64.9
	Test 2	59.9	1.26	38.9
	Test 3	56.0	0.64	43.4
	Average	50.3	0.66	49.0
	SD	13.4	0.60	13.9
SADS R	Test 1	63.6	1.73	34.6
	Test 2	59.3	0.48	40.2
	Test 3	53.0	0.75	46.3
	Average	58.6	0.99	40.4
	SD	5.4	0.66	5.8
Total Average		52.3	1.15	46.6
SD		5.6	0.58	5.4

669 **4.5 Detailed particles fate measured for SADS-R**

670 **Figure 10(a)** and **Figure 10(b)** present images depicting a typical deposition that occurs inside the SADS after the
 671 sampling process. Notably, a significant amount of liquid particles can be seen on the external walls of the nozzles.
 672 Deposits can also be found on the internal walls of the nozzles, but they are difficult to capture in photographs.
 673 Additionally, in certain tests, projections of macroscopic droplets from the nozzles to the walls of the plenum were
 674 also observed.



675 (a) acceleration nozzle (b) Collection nozzle

677 **Figure 10: Pictures of particles deposition outside the nozzle.**

678 To better understand the localization and underlying reasons for particle deposition in the SADS, independently
 679 of variations between prototypes, the detailed transmission ratios and position-resolved deposition ratios for the
 680 reference SADS-R are given on

681 **Figure 11**, for all tested particle sizes. Values are averaged over 3 repetitions.

682 For particles with reference diameter 0.15 μm , wall deposition is primarily located on the collection nozzle, with
 683 6 ± 5.8 % of sampled mass, though it should be almost zero theoretically (0.44 ± 0.58 %).

684 For particles with reference diameters 2, 3 and 4.5 μm , the experimental results show substantial particle
 685 depositions (43 % of sampled mass in average), unpredicted by theory either (0.2 ± 0.1 %). These experimental
 686 wall deposits seem almost independent of particle diameter for these inertial particles. The distribution of deposits

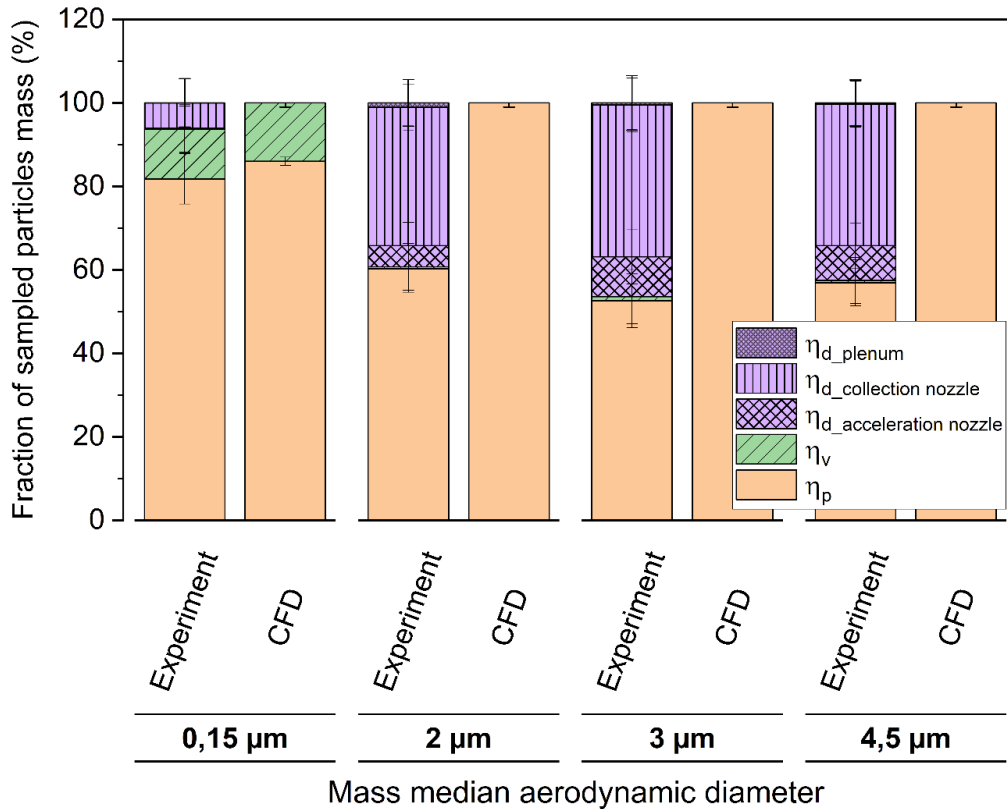
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691 across zones reveals that the majority of particles deposit on the collection nozzle ($34.5 \pm 3.4 \%$), followed by the
 692 acceleration nozzle ($7.6 \pm 3.4 \%$), with minimal deposition in the plenum ($0.6 \pm 3.4 \%$).



693

694 **Figure 11:** Mass distribution in SADS R exposed to four different particle size distributions: experimental and
 695 numerical study for precise wall deposition localization and transmission efficiency analysis. (Three repetitions, error
 696 bars represent measuring uncertainty). It is essential to note that most of the deposits observed can contaminate the
 697 vapour phase measurements at the minor outlet if particles evaporate after deposition. Indeed, wall deposition on the
 698 acceleration nozzle is located before separation, and most of the deposit of the collection nozzle is actually located on its
 699 outside walls (fraction of the deposit which is visible to the unaided eye).

700 Having ruled out the effects of leakage and machining inaccuracies in nozzle diameter, we can envisage several
 701 reasons for these deposits, which are not predicted by the numerical model. Firstly, the simulated geometry may
 702 not correspond to the real geometry for aspects other than nozzle diameter. In particular, the variations in deposits
 703 between the tests (Figure 9, and

704 Figure 11) suggest variability in the assembly of the 2 parts of the SADS in relation to each other, and therefore a
 705 geometry of the interior domain of the SADS that is not only variable but also different from what is simulated.
 706 These variations may correspond in particular to a misalignment of the nozzles with respect to each other, which
 707 can easily explain the impactation of inertial particles outside the collection nozzle (Loo & Cork, 1988). In the
 708 following section, the sensitivity of SADS performances with respect to nozzle misalignment is thus illustrated
 709 theoretically.

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714 **4.6 Theoretical effect of nozzle misalignment**

715 Study by Loo & Cork (1988) emphasized the importance of maintaining axial alignment between the acceleration
716 and collection nozzle of a VI. In their case, which is very different from the SADS in terms of dimensions and air
717 flow rates, they recommend avoiding an offset of more than 1.6 % of nozzle diameter and observe that each 1.6 %
718 increase in misalignment leads to a 1 % increase in nozzle wall loss. Meeting this criterion in the case of the SADS
719 would mean avoiding a misalignment of more than 0.013 mm, which is challenging from a mechanical design
720 point of view. Experimentally, measuring the misalignment offset of the mounted SADS was not feasible.
721 However, a sensitivity analysis can be performed by means of parametric CFD computations to explore the impact
722 of this parameter.

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724 Simulations were hence carried out with relative displacements of the collection nozzle with respect to the
725 acceleration nozzle (Figure 2) in the Z-axis direction, with likely values of the axial backlash between the two
726 parts of the SADS. These parts are assembled by manually fitting together a shaft and a 37 mm diameter hole.
727 Following ISO system of limits and fits, this corresponds to a H7/h6 clearance fit (location fit), whence a possible
728 axial backlash in the range 0 to 0.041 mm is deduced. Parametric computations were then performed for axial
729 backlashes of 0, 0.025 and 0.041 mm respectively (corresponding to 0, 3.1 % and 5.1 % of the nozzle diameter
730 respectively). A simulation with an extreme backlash of 0.075 mm (9.4 % of nozzle diameter) was also performed
731 for information, which could correspond to a more tolerant H8/f7 ISO clearance fit (close-running fit).

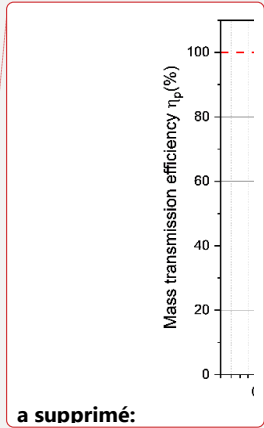
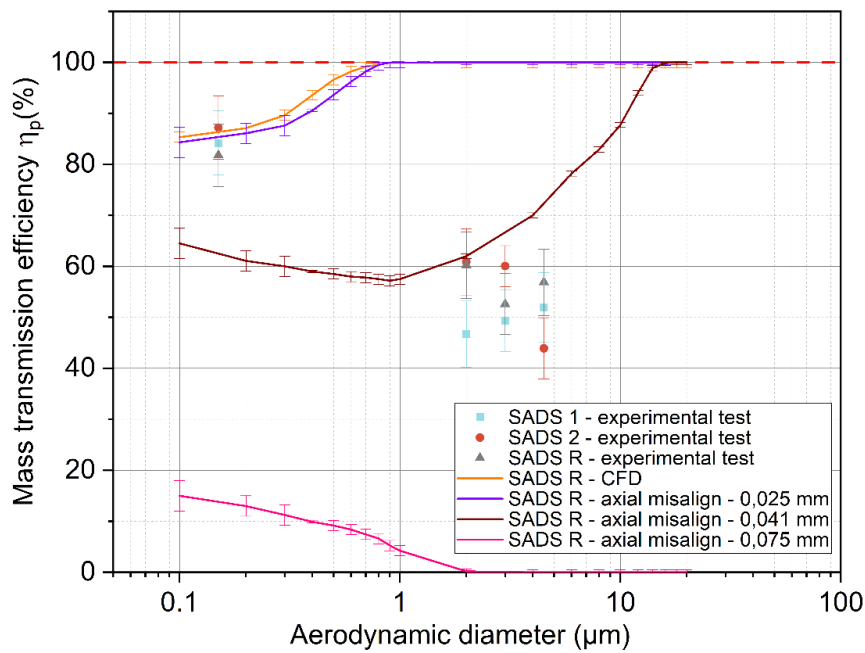
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732 Figure 12 displays the corresponding numerical simulation results for the transmission efficiency and wall
733 deposition, compared with experimental measurements of SADS 1, 2, and R, illustrating the possible effect of
734 axial misalignment on η_p .

a supprimé: Figure 11

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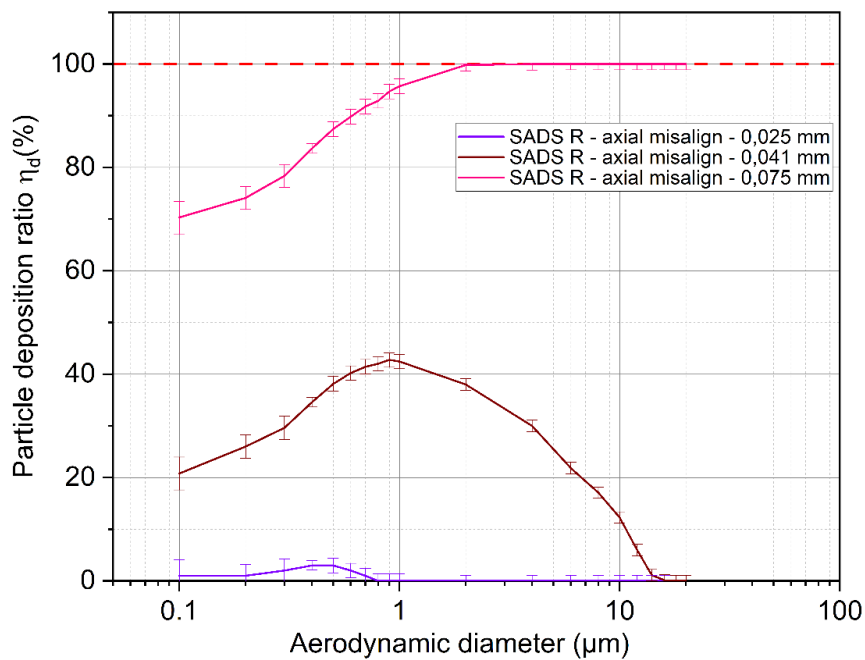
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A)



742

B)

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744

Figure 12: Misalignment effect on mass transmission efficiency (A) and Particle deposition ration (B), evaluated by numerical simulations and experimental tests.

745
746
747
748

Results show that likely values of the axial mechanical backlash between the two parts lead to a severe decrease of η_p due to dramatically increasing wall losses η_d on the collection nozzle external walls, especially for the most inertial particles. This finding is compatible with experimental measurements. Axial misalignment of the device is therefore a possible cause of the differences in performance between the ideal version and the mechanical

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758 realization of the SADS, for the most inertial particles tested. Of course, present simulations can only qualitatively
759 reproduce the tendency of the experiment, since they were not performed with the actual axial backlash which is
760 unknown and which varies between each SADS assembly. Also, the effects of other existing mechanical
761 backlashes were not numerically tested and necessarily contribute to sampling performances (tolerance on the
762 separation length between nozzles, existing angle between the axes of the two parts of the SADS, etc.).
763 Based on the analysis of the results, we can conclude that the maximum allowable misalignment during the
764 assembly of the SADS, between the collection nozzle and the acceleration nozzle, is established at 0.025 mm, as
765 evidenced by the violet curve. Notably, this curve consistently aligns with that of the SADS R-CFD, representing
766 a curve with perfect alignment.

767 **5 Conclusion and discussion**

768 This study experimentally evaluated the dichotomous sampling performance (gas and particles) of 5 SADS
769 prototypes (2009 version) of identical design, and for an aerosol of liquid particles with aerodynamic diameters of
770 0.15, 2, 3 and 4.5 μm . The study was carried out for constant air flows set in accordance with SADS specifications.
771 Lab tests were carried out in a dedicated controlled generation environment, the BAO, in which monodisperse
772 aerosols marked with a fluorescent dye were emitted, thanks to the modification of the CMAG generator to
773 accommodate the use of glycerol and fluorescein as condensing vapour and nuclei respectively.

774 SADS sampling performance in terms of total mass sampled, particle fraction transmitted to the major outlet and
775 particle losses at the walls were put into perspective with the details of mechanical construction and with the
776 operating conditions of the prototypes: leak rate, repeatability between successive assemblies, imprecision of
777 machined nozzles diameters, and axial misalignment of the assembly.

778 The measured sampling performances were compared to their theoretical counterparts computed by CFD in Belut
779 et al. (2022), and CFD was also used to study the theoretical effect of variations in the geometry of machined
780 SADS relative to the plans, regarding the nozzles diameters and nozzle misalignments.

781 With the originally proposed design, the SADS assembly was found easily leaky, but consequences on sampling
782 could be overcome when a prior leak test with leakage rate LR below 4 Pa.s⁻¹ was passed.

783 Sampling performances were found little reproducible during successive SADS assemblies (between tests,
784 CV=22.1 % for wall losses). Theoretically unpredicted large (40-46 % of sampled mass) wall losses were
785 measured for particles larger than 2 μm , located mostly (80 %) on the external walls of the collection nozzle.
786 Assembly repeatability issues and simulations of SADS parts misalignment effect by CFD suggest that these
787 undesirable particle deposits are due to the mechanical backlashes of the assembly, and not to the imprecision of
788 actually machined nozzles diameters. Indeed, the measured variation of nozzle diameters in the range (-4.5 %,
789 +3.7 %) with respect to nominal values were found to theoretically affect marginally (<3 %) aerosol transmission
790 efficiencies.

791 Present results suggest that although the dichotomous sampling performances of the SADS are theoretically
792 interesting for workplace exposure assessment to SVOC aerosols, its actual realization fails in reaching theoretical
793 performances for micron-sized particles, due to mechanical design issues. Airtightness, nozzle alignment and
794 repeatability of assembly are not sufficiently guaranteed by its initial design and future development should focus
795 on improving these aspects to obtain a sampler suitable for field studies.

796 However, several biases may have affected the findings of the study. For example, rare macroscopic particles are
 797 sometimes emitted by the CMAG and may have been sampled by the SADS, biasing the separation performance
 798 measurements and especially the deposition measurements. However, we believe that this possibility is largely
 799 controlled by the real-time monitoring of particle sizes in the test rig, and by the simultaneous use of several SADS
 800 in the test rig. Regarding the plausibility of the simulation results, it is, of course, limited by the physical
 801 phenomena actually taken into account. Calculation errors are limited by the verification procedure used (Belut et
 802 al., 2022), which guarantees a numerical error of less than 0.5 % on the particulate fractions deposited and
 803 transmitted. However, actual variations in the geometry of assembled SADS compared with the drawings (due to
 804 machining inaccuracies other than nozzle diameters) are not taken into account, nor is wall roughness, despite its
 805 acknowledged effect on deposits. The roughness of the machined acceleration nozzles could therefore help to
 806 explain the deposition of particles in this nozzle, which are not predicted by calculations that assume a perfectly
 807 smooth nozzle. Similarly, the more or less pronounced sharpness of the sampling orifice actually machined can
 808 have a significant influence on the inlet particle velocity and concentration profile, and therefore on the actual
 809 performance of the SADS (Belut et al., 2022).

810 Nevertheless, the lack of repeatability of SADS performance between successive assemblies, its low and variable
 811 airtightness level and its radial clearance large enough to cause a significant misalignment of the nozzles (typically
 812 5 %) have sufficiently important effects for these possible limitations of the study not to call into question its
 813 conclusions.

814 Overall, these results clearly show that it is mechanically difficult to design a SADS that meets the theoretical
 815 specifications. In fact, the alignment tolerances require precise machining, which may be an obstacle to the
 816 development of this device. It should be added that the head losses of the device at its nominal flow rate are
 817 1400 Pa on the major flow side and 3700 Pa on the minor flow side (Belut et al., 2022). These head losses are at
 818 the limit of the performance of individual sampling pumps, especially when considering the additional head losses
 819 caused by the collection media downstream of the SADS outlets. This raises the question of whether the device
 820 should be completely redesigned, with larger nozzle diameters that are easier to align mechanically and generate
 821 less pressure drop.

822 By addressing the identified challenges and incorporating further refinements in the SADS design and operation,
 823 researchers can enhance its reliability, accuracy, and applicability in various aerosol sampling applications,
 824 contributing to advancements in aerosol science and related fields.

a supprimé:
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a supprimé: Add here comment about SADS pressure drops and improvement possibilities (larger diameters as pointed out by rev#2)

825 **7 Table of Symbols**

<u>Greek Letters</u>		
Δt	<u>Duration of the leak test</u>	(s)
η_p	particles transmission efficiency to the particle outlet (major flow)	(-)
η_v	particles transmission efficiency to the vapour outlet (minor flow)	(-)
η_d	particles deposition ratio	(-)

$\eta_{da}, \eta_{dc}, \eta_{dp}$	particles deposition ratio in the acceleration nozzle, collection nozzle and in the plenum respectively	(-)
μ	<u>Air viscosity</u>	<u>Pa.s⁻¹</u>
ρ	<u>Air density</u>	<u>kg.m⁻³</u>
ρ_0	<u>Reference particle density, equals to 1000 kg.m⁻³</u>	<u>(kg.m⁻³)</u>
ρ_p	<u>Relative particle density</u>	<u>(kg.m⁻³)</u>
x	<u>Shape factor</u>	(-)
<u>Lowercase Latin letters</u>		
d_{ae}	<u>Aerodynamic diameter</u>	<u>(μm)</u>
d_m	<u>Electrical mobility diameter</u>	<u>(μm)</u>
d_{ev}	<u>Equivalent volume diameter</u>	
m_{CFC}	<u>Mass of particles collected inside a close-face cassette</u>	<u>(ng)</u>
m_{depa}	<u>Mass of particles collected on the acceleration nozzle walls of the SADS</u>	<u>(ng)</u>
m_{dep_c}	<u>Mass of particles collected on the collection nozzle walls of the SADS</u>	<u>(ng)</u>
m_{dep_p}	<u>Mass of particles collected on the plenum walls of the SADS</u>	<u>(ng)</u>
m_{inlet}	<u>Mass of sampled particles at the inlet</u>	<u>(ng)</u>
m_{major}	Mass of particles collected at the major flow outlet	(ng)
m_{minor}	Mass of particles collected at the minor flow outlet	(ng)
<u>Uppercase Latin letters (Variables)</u>		
C_u	<u>the Cunningham slip correction factor</u>	(-)
GSD	<u>Geometric standard deviation</u>	(-)
$LLOQ$	<u>Lower limit of quantification</u>	<u>(ng.L⁻¹)</u>
LR	<u>Leakage rate</u>	<u>(Pa.s⁻¹)</u>
Mb_{SADS}	<u>Mass balance of SADS</u>	(-)
$MMAD$	<u>Mass median aerodynamic diameter</u>	<u>(μm)</u>
P	<u>Pressure</u>	<u>(Pa)</u>
P_0 and P_f	<u>Pressures inside a SADS at T=0 and T=final time, during the leak test.</u>	<u>(Pa)</u>
T	<u>Temperature</u>	<u>(°C)</u>
$ULOQ$	<u>Upper limit of quantification</u>	<u>(ng.L⁻¹)</u>
V_p	<u>Particle volume</u>	<u>(μm^3)</u>
<u>Abbreviations</u>		
<u>APS</u>	<u>Aerodynamic Particle Sizer</u>	
<u>BOA</u>	<u>Bench of Organic Aerosol</u>	
<u>CFC</u>	<u>Close-Face cassette</u>	

a supprimé: acceleration

a supprimé: collection

a supprimé: nozzle

a supprimé: m_{inlet} ...

a supprimé: Mb_{SADS} Mass balance of SADS (-)

<u>CMAG</u>	<u>Condensation Monodisperse Aerosol Generator</u>	
<u>FMPS</u>	<u>Fast Mobility Particle Sizer</u>	
<u>OFC</u>	<u>Open-Face cassette</u>	
<u>SADS</u>	<u>Semivolatile Aerosol Dichotomous Sampler</u>	
<u>SEM</u>	<u>Scanning Electron Microscopy</u>	
<u>WD</u>	Wall depositions	

a supprimé: MMAD

a supprimé: m_{WL}

840 8. Author contribution

841 BS, EB, EG and RO conceptualized the research project. NR, BS and EB developed the methodology and NR was
842 responsible of the investigation. BS and EB validated the results. EG was responsible for the supervision of the
843 project. NR wrote the original draft preparation and BS, EB and EG reviewed and edited the manuscript.

844 9. Competing interests

845 The authors declare that they have no conflict of interest.

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